

Energy-Momentum-Consistent Simulation of Planar Geometrically Exact Beams as PHS

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Motivation

Flexible multibody systems (MBS)

Wishlist for a formulation:

- Allow finite rotations and displacements
- Avoid numerical *locking*
- Robust and objective
- Energy-momentum (EM) consistent discretization

PHS modeling ideally suited:

- Structured formalism: Energetic consistency
- Intrinsically mixed
- Modular interconnection
- Easily extendable: DAEs and multiphysics

Problem description

- Geometrically exact beams (GEB)
 - IBVP on a 1D continuum with $s \in \Omega = [0, L]$

- Nonlinear 2D kinematics:

- Position $\mathbf{r}(s, t) \in \mathbb{R}^2$ and orientation $\varphi(s, t)$
- Rotation matrix

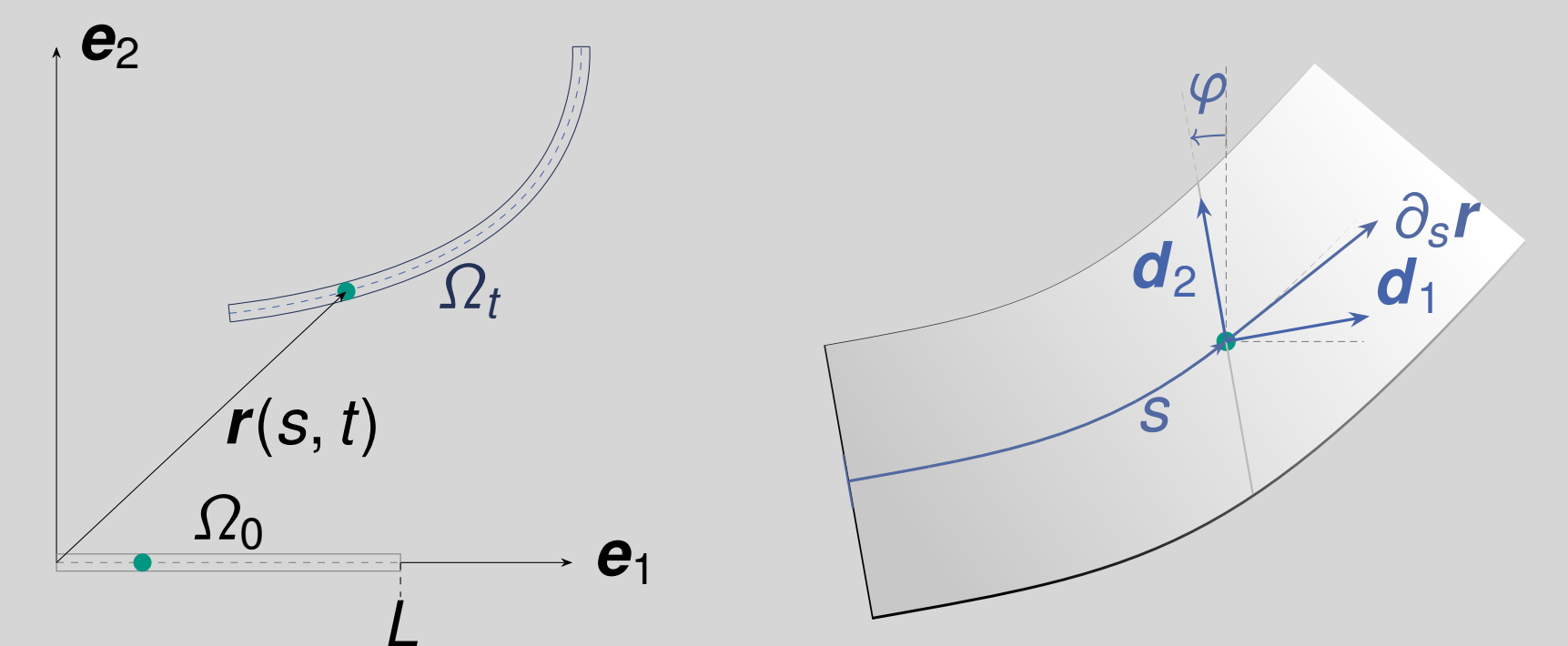
$$\mathbf{A}(\varphi) = \begin{bmatrix} \cos \varphi & -\sin \varphi \\ \sin \varphi & \cos \varphi \end{bmatrix} = [\mathbf{d}_1(\varphi) \ \mathbf{d}_2(\varphi)] \in SO(2)$$

- Strains (dilatation, shear and curvature)

$$\boldsymbol{\Gamma} = \mathbf{A}^\top \partial_s \mathbf{r} - \mathbf{e}_1 \in \mathbb{R}^2 \quad \kappa = \partial_s \varphi$$

- Internal forces, moments and strain energy function

$$\mathbf{n} = \mathbf{A} \mathbf{N} \quad m = M \quad \mathbf{N} = \partial_{\boldsymbol{\Gamma}} W \quad M = \partial_{\kappa} W$$



- Kinematic evolution

$$\partial_t \mathbf{r} = \mathbf{v} \quad \partial_t \varphi = \omega \quad \partial_t \kappa = \partial_s \omega \quad \partial_t \boldsymbol{\Gamma} = \omega \mathbf{A}^\top \mathbf{S} \partial_s \mathbf{r} + \mathbf{A}^\top \partial_s \mathbf{v}$$

- Balance equations with external loads $(\bar{\mathbf{n}}, \bar{m})$

$$\rho A \partial_t \mathbf{v} = \partial_s \mathbf{n} + \bar{\mathbf{n}} \quad \rho I \partial_t \omega = \partial_s M + \partial_s \mathbf{r}^\top \mathbf{S} \mathbf{n} + \bar{m}$$

PH modeling and structure-preserving discretization in space and time

PH formulation

- State (energy-variables)

$$\mathbf{x} = (\mathbf{r}, \varphi, \mathbf{v}, \omega, \boldsymbol{\Gamma}, \kappa)$$

- Hamiltonian

$$\mathcal{H}(\mathbf{x}) = \int_{\Omega} \left(\frac{1}{2} \rho A \mathbf{v}^\top \mathbf{v} + \frac{1}{2} \rho I \omega^2 + W(\boldsymbol{\Gamma}, \kappa) \right) ds$$

- Local PH equations (where $\mathcal{J} = -\mathcal{J}^*$)

$$\mathcal{E} \partial_t \mathbf{x} = \mathcal{J}(\mathbf{x}) \mathbf{z}(\mathbf{x}) + \mathbf{B} \mathbf{u} \quad \text{with} \quad \mathcal{E}^\top \mathbf{z}(\mathbf{x}) = \delta_{\mathbf{x}} \mathcal{H} \quad \mathbf{y} = \mathbf{B}^\top \mathbf{z}(\mathbf{x})$$

- Power balance

$$\partial_t \mathcal{H} = [\mathbf{v}^\top \mathbf{n} + \omega M]_{\partial \Omega} + (\mathbf{u}, \mathbf{y})_{\Omega} = \mathbf{u}_{\partial}^\top \mathbf{y}_{\partial} + (\mathbf{u}, \mathbf{y})_{\Omega}$$

Mixed FEM

- 2-node elements, ansatz functions:

- C^0 -continuous, linear for $(\mathbf{r}, \mathbf{v}, \varphi, \omega)$
- discontinuous and constant for $(\boldsymbol{\Gamma}, \kappa)$

- Discrete state $\hat{\mathbf{x}} = (\hat{\mathbf{r}}, \hat{\varphi}, \hat{\mathbf{v}}, \hat{\omega}, \hat{\boldsymbol{\Gamma}}, \hat{\kappa})$ and

$$\hat{\mathcal{H}}(\hat{\mathbf{x}}) = \frac{1}{2} \hat{\mathbf{v}}^\top \rho A \mathbf{M}_2 \hat{\mathbf{v}} + \frac{1}{2} \hat{\omega}^\top \rho I \mathbf{M}_1 \hat{\omega} + \hat{W}(\hat{\boldsymbol{\Gamma}}, \hat{\kappa})$$

- Resulting finite-dimensional PHS

$$\mathbf{E} \partial_t \hat{\mathbf{x}} = \mathbf{J}(\hat{\mathbf{x}}) \hat{\mathbf{z}}(\hat{\mathbf{x}}) + \mathbf{B} \hat{\mathbf{u}} \quad \text{with} \quad \mathbf{E}^\top \hat{\mathbf{z}}(\hat{\mathbf{x}}) = \nabla_{\hat{\mathbf{x}}} \hat{\mathcal{H}} \quad \hat{\mathbf{y}} = \mathbf{B}^\top \hat{\mathbf{z}}(\hat{\mathbf{x}})$$

- Power balance: $\partial_t \hat{\mathcal{H}} = \hat{\mathbf{u}}^\top \hat{\mathbf{y}}$

Midpoint-based integration

- 2nd-order one-step schemes with $\mathbf{x}_n \approx \mathbf{x}(t_n)$, $h = \text{const.}$ and $\hat{\mathbf{x}}_{n+1/2} = \frac{1}{2}(\hat{\mathbf{x}}_{n+1} + \hat{\mathbf{x}}_n)$ given by

$$\mathbf{E}(\hat{\mathbf{x}}_{n+1} - \hat{\mathbf{x}}_n) = h \mathbf{J}(\hat{\mathbf{x}}_{n+1/2}) \hat{\mathbf{z}}_{n,n+1} + h \mathbf{B} \hat{\mathbf{u}}_{n,n+1} \quad \hat{\mathbf{y}}_{n,n+1} = \mathbf{B}^\top \hat{\mathbf{z}}_{n,n+1}$$

- Discrete-time PHS by discrete gradient

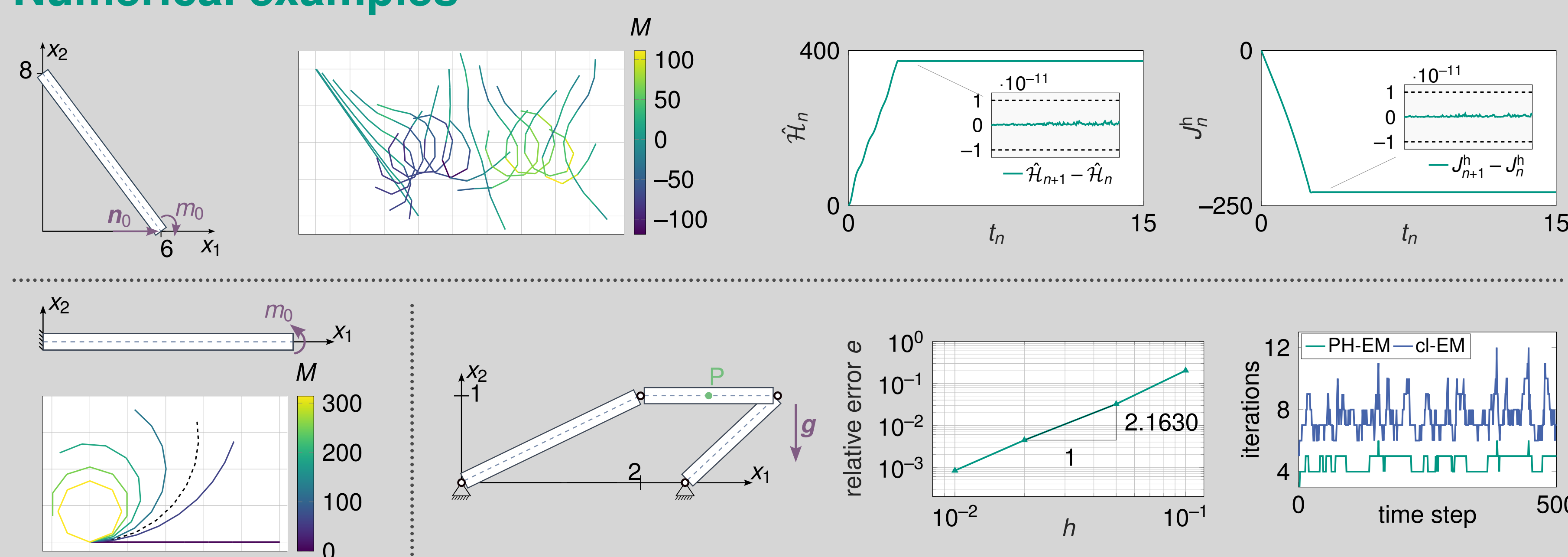
$$\mathbf{E}^\top \hat{\mathbf{z}}_{n,n+1} = \nabla \hat{\mathcal{H}}(\hat{\mathbf{x}}_n, \hat{\mathbf{x}}_{n+1})$$

- Consistent power balance

$$\hat{\mathcal{H}}_{n+1} - \hat{\mathcal{H}}_n = h \hat{\mathbf{u}}_{n,n+1}^\top \hat{\mathbf{y}}_{n,n+1}$$

- Consistent angular momentum balance

Numerical examples



Take-Home message

We achieved a combination of two state-of-the-art modeling approaches:

- GEB:** Finite displacements and rotations ✓
- PHS:** Mixed, objective and robust approach yields EM-scheme in a natural way and avoids locking ✓

Together they provide a powerful tool for accurate simulations of flexible MBS

Future research directions:

- Interconnection with other PH MBS models
- Extension to 3D (rotation parametrization)

References

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- PLK, T. Thoma, PB, P. Kotyczka (2024): Generalized Maxwell viscoelasticity for geometrically exact strings. In: IFAC-PapersOnLine 58.6, doi: [10.1016/j.ifacol.2024.08.264](https://doi.org/10.1016/j.ifacol.2024.08.264)
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